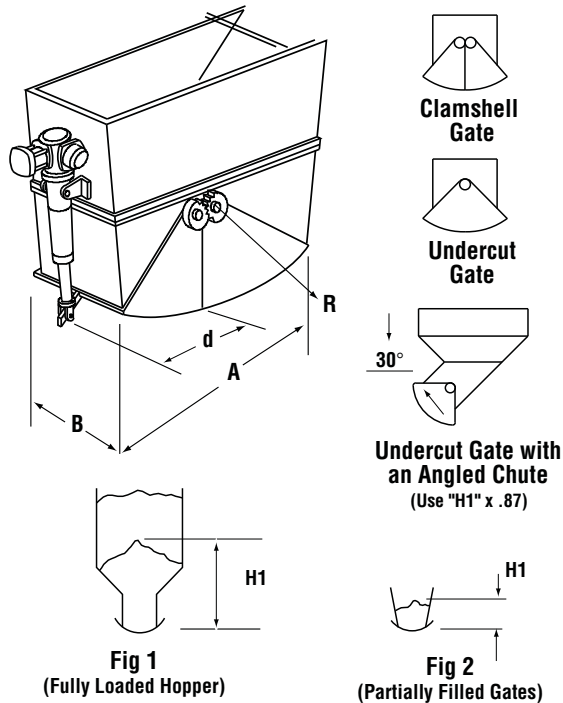


# GENERAL APPLICATIONS

## CLAM SHELL OR UNDERCUT GATES



**Gate Operation:** The maximum force required to operate a clam shell and undercut gates occur when opening from a fully closed position when material is resting on the gate blade. To move the gate, static friction ( $\mu_2$ ) must be overcome. Once moving, the material weight on the gate is reduced, the coefficient of friction is lower and the required force is reduced proportionally as the gate opens.

**Required Information:** Dimensions A, B, d\* and R,\* material being handled, gate weight (W3) and approximate material height above the gate blade "H1".  
\*See legend if dimensions are not known.

### Legend

A	length of gate opening (ft)
B	width of gate opening (ft)
d	effective length of lever arm (ft) (If specific dimension is not known use length of gate "A"/1.9)
H1	effective height of material on the gate blade (ft). See Figure 1 and 2
R	radius of gate (ft). If specific dimension is not known use (A/2 x 1.5).
W1	average material specific weight (lbs/ft <sup>3</sup> ) (Table 2)
W2	material weight on gate blade (lbs).
W3	gate weight (lbs).
$\mu_2$ & $\mu_4$	coefficients of starting ( $\mu_2$ ) and running ( $\mu_4$ ) friction between the material in the hopper and the gate blade. (Table 2)
a	acceleration factor (Table 3)
Baf	required breakaway force (lbs)
Rf (avg)	average required running force (lbs)
Fig 1	Material in a hopper will start to bridge or support itself a certain height (H1) above the gate blade. To find "H1" use the "h" factor from Table 2, and the formula.
Fig 2	If the actual material above the gate blade is less than the calculated value of "H1" use the actual height in feet.

Reference Tables on Page 45.

### Formulas

$$H1 = \left[ \begin{array}{l} \text{Larger of A or B} \\ \text{dimensions} \end{array} \right] \times \left[ \begin{array}{l} \text{"h" factor from} \\ \text{Table 2} \end{array} \right] = \text{_____ ft}$$

$$W2 = [A \text{ ft} \times B \text{ ft} \times H1 \text{ ft}] \times W1 \text{ (lbs/ft}^3\text{)} = \text{_____ lbs}$$

$$Baf = \frac{[(W2 \text{ lbs} \times \mu_2) + W3 \text{ lbs}] \times R \text{ (ft)} \times a}{d \text{ (ft)}} = \text{_____ lbs}$$

$$Rf(\text{avg}) = \frac{[(W2 \times \mu_4) + W3 \text{ lbs}] \times R \text{ (ft)}}{d(\text{ft}) \times 1.75} = \text{_____ lbs}$$

### Example

Determine the actuator force required to open a 4 ft. square coal refuse clam shell gate. The total weight of the gate is 750 lbs. No other dimensions are known.

$$W2 = 4 \text{ ft} \times 4 \text{ ft} \times (4 \text{ ft} \times 2.0) \times 45 \text{ (lbs/ft}^3\text{)} = 5760 \text{ lbs}$$

$$W3 = 750 \text{ lbs}, R = (4 \text{ ft}/2 \times 1.5) = 3 \text{ ft}, d = 4 \text{ ft}/1.9 = 2.11 \text{ ft}$$

$$Baf = \frac{[(5760 \text{ lbs} \times .75) + 750 \text{ lbs}] \times 3 \text{ ft} \times 1.2}{2.11 \text{ ft}} = 8650 \text{ lbs}$$

$$Rf(\text{avg}) = \frac{[(5760 \text{ lbs} \times 0.5) + 750 \text{ lbs}] \times 3 \text{ ft}}{2.11 \text{ ft} \times 1.75 \text{ ft}} = 2950 \text{ lbs}$$

\* Acceleration factor based on velocity of 3.5 in/sec or less.

### APPLICATION NOTES

1. Select actuator according to breakaway force requirements.

# ENGINEERING TABLES

**TABLE 1**

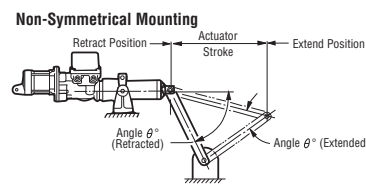
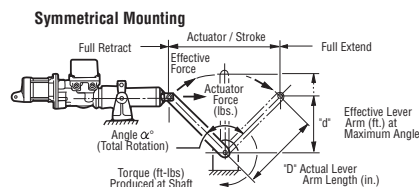
Guide Type	Coefficient of Friction Between Steel Plate and Various Type Guides			
	Dry		Lubricated	
	Starting ( $\mu_1$ )	Running ( $\mu_3$ )	Starting ( $\mu_1$ )	Running ( $\mu_3$ )
Ball Bearings	–	–	0.08	0.05
Steel Rollers	–	–	0.15	0.10
Bronze	0.45	0.20	0.2	0.15
Steel	0.4-0.8	0.2-0.4	0.2-0.4	0.15-0.2

**TABLE 2**

Material	Coefficient of Friction Between Steel Plate and Various Materials		Specific Weights "W1" (lbs/ft <sup>3</sup> )	Column Height Factor "h"
	Starting ( $\mu_2$ )	Running ( $\mu_4$ )		
Ash (fly)	0.6-0.7	0.3	40-45	3.0
Ash (wet) (coal refuse)	0.75-0.95	0.5	45-50	2.0
Cement (portland)	0.6-0.65	0.3	95-100	3.0
Cement (clinker)	0.55-0.6	0.3	80-95	4.0
Coal (anthracite)	0.50-0.55	0.25	55-60	4.5
Coal (bituminous)	0.55-0.6	0.3	45-55	3.0
Coke	0.55-0.6	0.3	25-35	3.5
Grain	0.32-0.40	0.2	40-50	4.75
Iron Ore	0.55-0.65	0.3	125-180	3.5
Limestone (crushed)	0.55-0.65	0.3	80-90	3.5
Rock (crushed)	0.65-0.7	0.3	125-140	4.0
Sand (dry)	0.5-0.55	0.3	90-110	4.0
Sand (damp)	0.6-0.65	0.4	110-125	2.5
Slag (blast furnace)	0.4-0.45	0.2	80-90	5.5
Steel	See Table 1	See Table 1	490	–
Taconite	0.35-0.4	0.2	120-130	8.25
Wood chips	0.75-0.8	0.4	10-30	2.5

**TABLE 3**

Velocity (in/sec.)	0.1 to 3.5	3.6 to 6.4	6.4 to 12.2	12.3 to 25.0
Acceleration Factor (a)	1.2	1.3	1.4	1.5



**TABLE 4**

Angle a	6" Stroke		12" Stroke		18" Stroke		24" Stroke	
	d'	D"	d'	D"	d'	D"	d'	D"
30°	.93	11.6	1.87	23.2	2.80	34.8	3.73	46.4
45°	.60	7.8	1.21	15.7	1.81	23.5	2.41	31.4
60°	.43	6.0	.87	12.0	1.30	18.0	1.73	24.0
90°	.25	4.2	.50	8.5	.75	12.7	1.00	17.0

Angle a	30" Stroke		36" Stroke		48" Stroke		60" Stroke	
	d'	D"	d'	D"	d'	D"	d'	D"
30°	4.67	58.0	5.60	69.5	7.46	92.7	9.33	115.9
45°	3.02	39.2	3.62	47.0	4.83	62.7	6.04	78.4
60°	2.17	30.0	2.60	36.0	3.46	48.0	4.33	60.0
90°	1.25	21.2	1.50	25.5	2.00	33.9	2.50	42.4

Angle $\theta^\circ$	5°	15°	30°	45°	60°	75°	90°
Force Factor	0.09	0.26	0.50	0.71	0.87	0.97	1.00

Effective force (lbs) = Actuator force (lbs) x force factor  
 Effective lever arm (ft) = Actual length arm (ft) x force factor

**NOTES**

1. For non-symmetrical mounting the angle  $\theta^\circ$  will change as the actuator moves through its travel. The angle  $\theta^\circ$  is formed between the actuator drive rod centerline and the lever arm axis.

Torque (ft/lbs) = Effective lever arm (ft) x actuator force (lbs)  
 Force Required (lbs) = Torque (ft/lbs) ÷ Effective lever arm (ft)